

Respect the Rheology!

Thickened Biosolids Pumping for Beneficial Reuse in Columbus, Ohio
December 7, 2017

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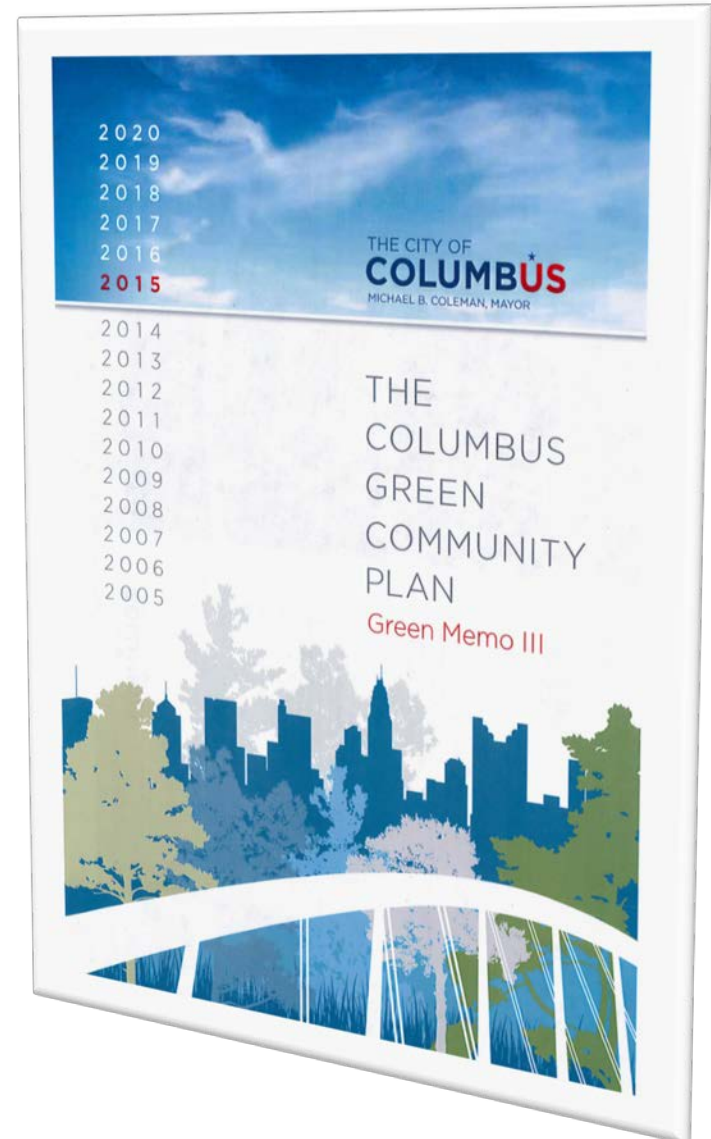
Agenda

- Overview of Columbus Biosolids Land Application Program/Project
- Sludge Pumping Design Practices
- Rheology 101
- Southerly WWTP Biosolids Land App Project
 - Field Testing
 - Rheology Testing
- Results and Startup

Overview of Columbus Biosolids Land Application Program

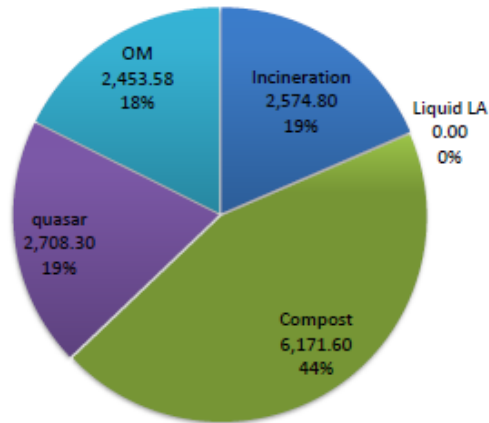
Columbus Sustainability Goals

- City sustainability goals described in the “Green Memo”
- Focus on conservation, efficiency, and renewables/reuse
- “Eliminate use of incinerators at Southerly WWTP...”

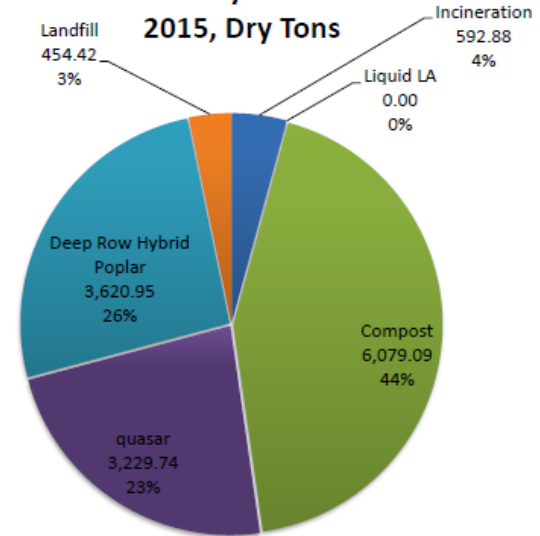


Southerly WWTP Solids Disposal

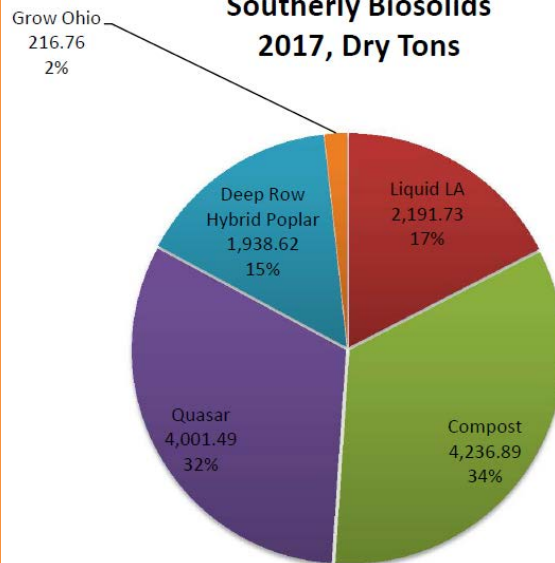
**Southerly Biosolids Distribution
2014, Dry Tons**



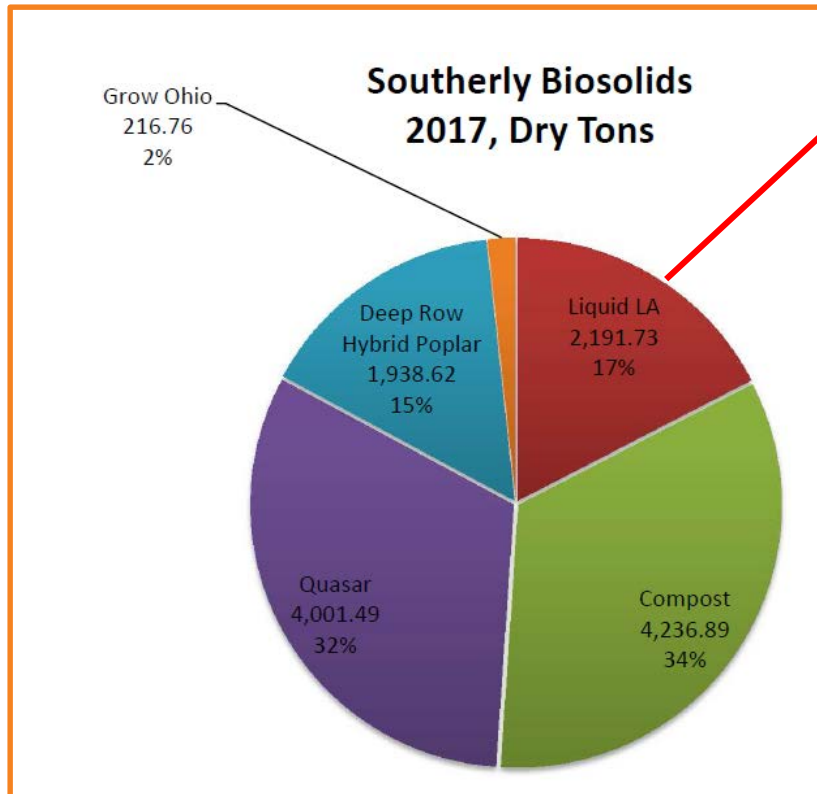
**Southerly Biosolids
2015, Dry Tons**



**Southerly Biosolids
2017, Dry Tons**



SWWTP: 2017 (data through November)



- Liquid Land Application
 - 9,070,000 gallons
 - 37,400 wet tons
 - In Storage – 96,000 gallons

Biosolids Land Application

- Overview of BLAF project
 - 8 Mgal of storage for thickened biosolids
 - Truck loading station
 - Pump/control building for truck load out
 - Repurposed centrifuges/new thickened biosolids pumps
- Thickened biosolids:
 - Centrifuge thickened with polymer addition to 8-10% TS.



Truck Loading Station



BLAF Facilities

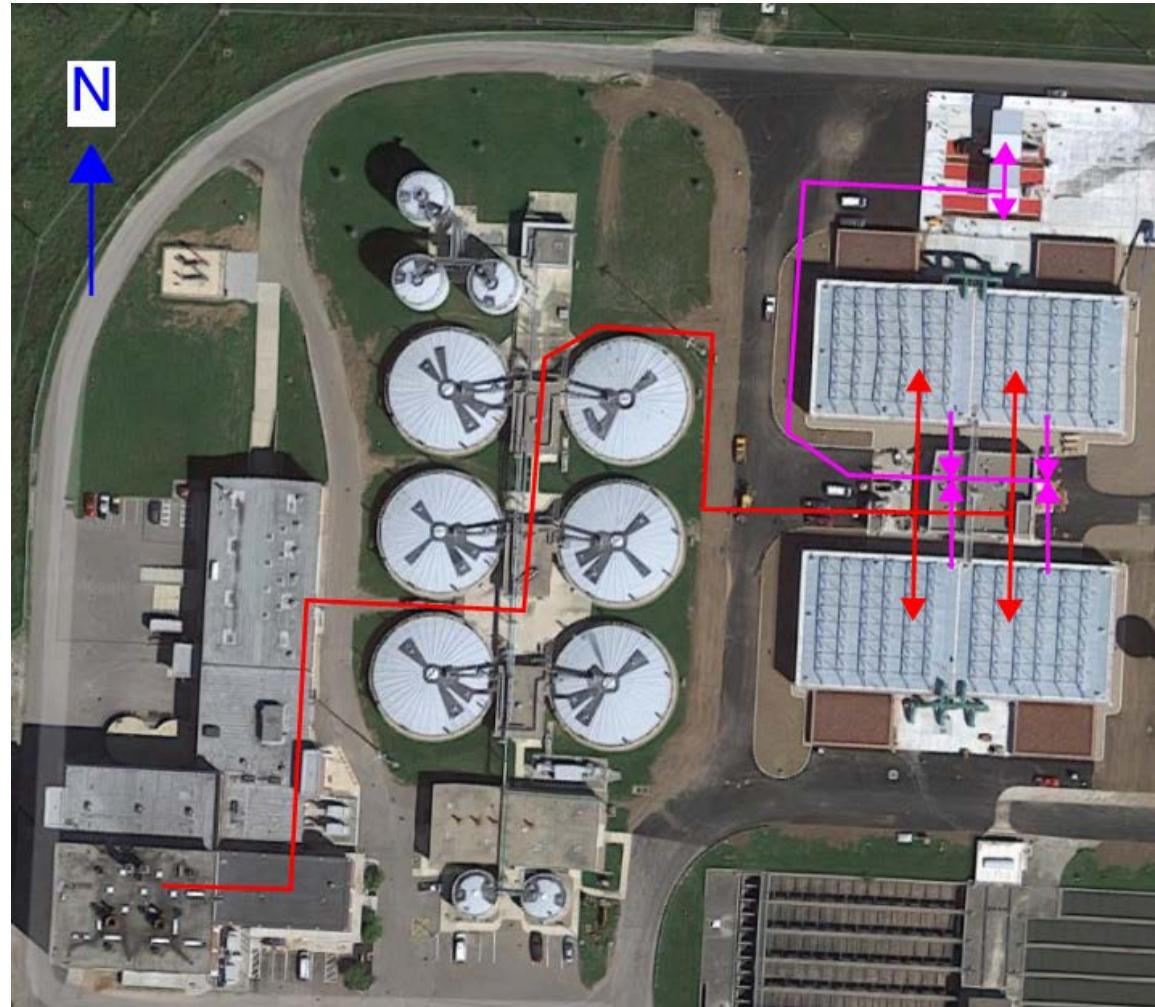
Two Thickened Biosolids (TBS) Pumping Systems Designed

TBS Transfer Pump

- From centrifuges to storage tanks

Truck Loading/Recirculation

- From tanks to truck loading, tank recirculation



The 8-million gallon question:

- How are we going to move this stuff (10%TS Biosolids)?
 - Not dry/typical cake %TS
 - Very thick (for a “liquid”), non-Newtonian characteristics.

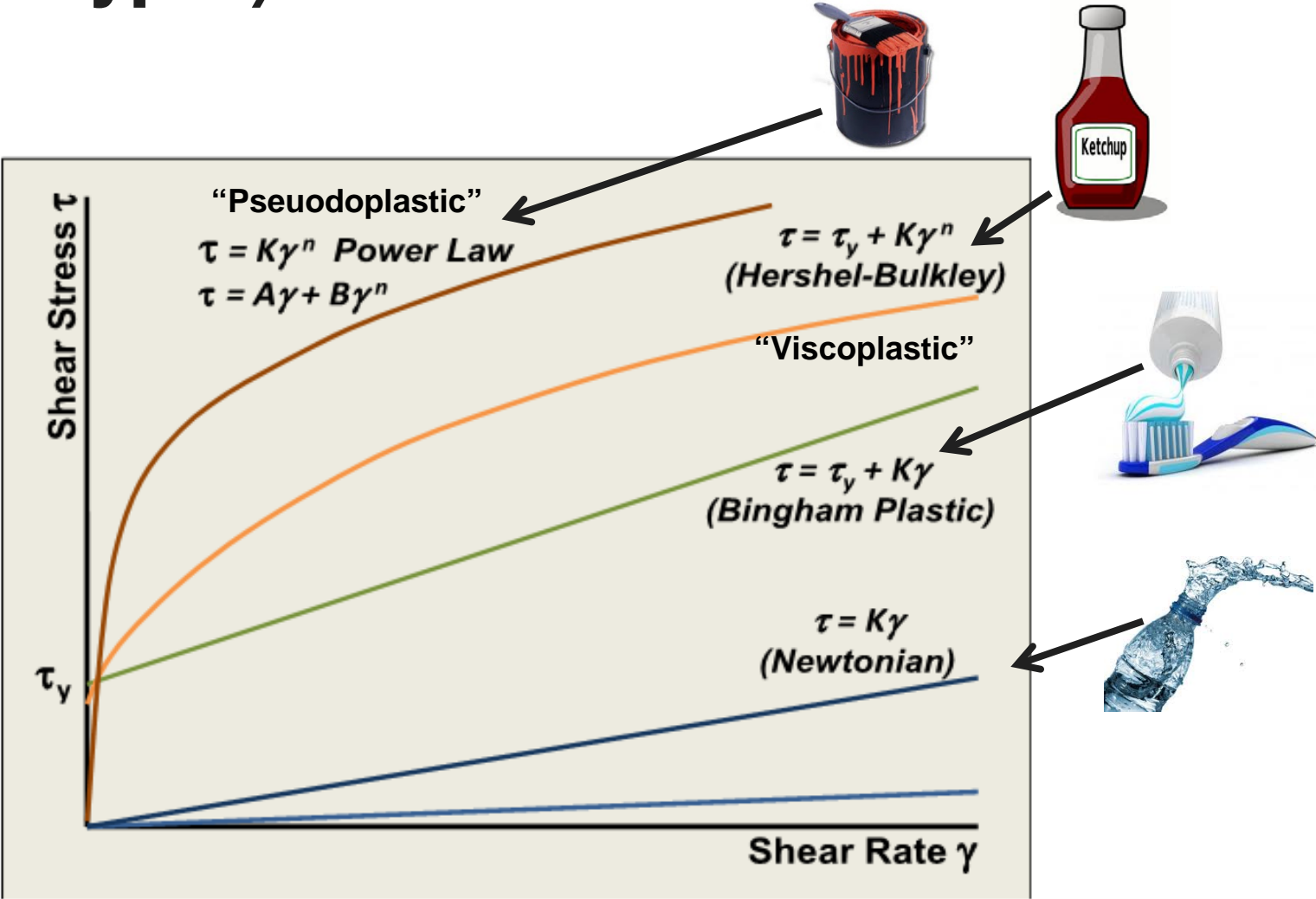
The 8-million gallon question:



First... a little Rheology 101

- Newtonian Fluid: viscosity constant, independent of shear rate (water).
- Non-Newtonian: viscosity not constant with change in shear rate (Literature says wastewater sludge once solids concentration $> \sim 2\%$).
- Shear-thinning: viscosity decreases as shear-rate increases.
- Yield Stress: minimum amount of force (shear) applied to initiate flow.
- Thixotropy: viscosity decreases over time when constant shear is applied.

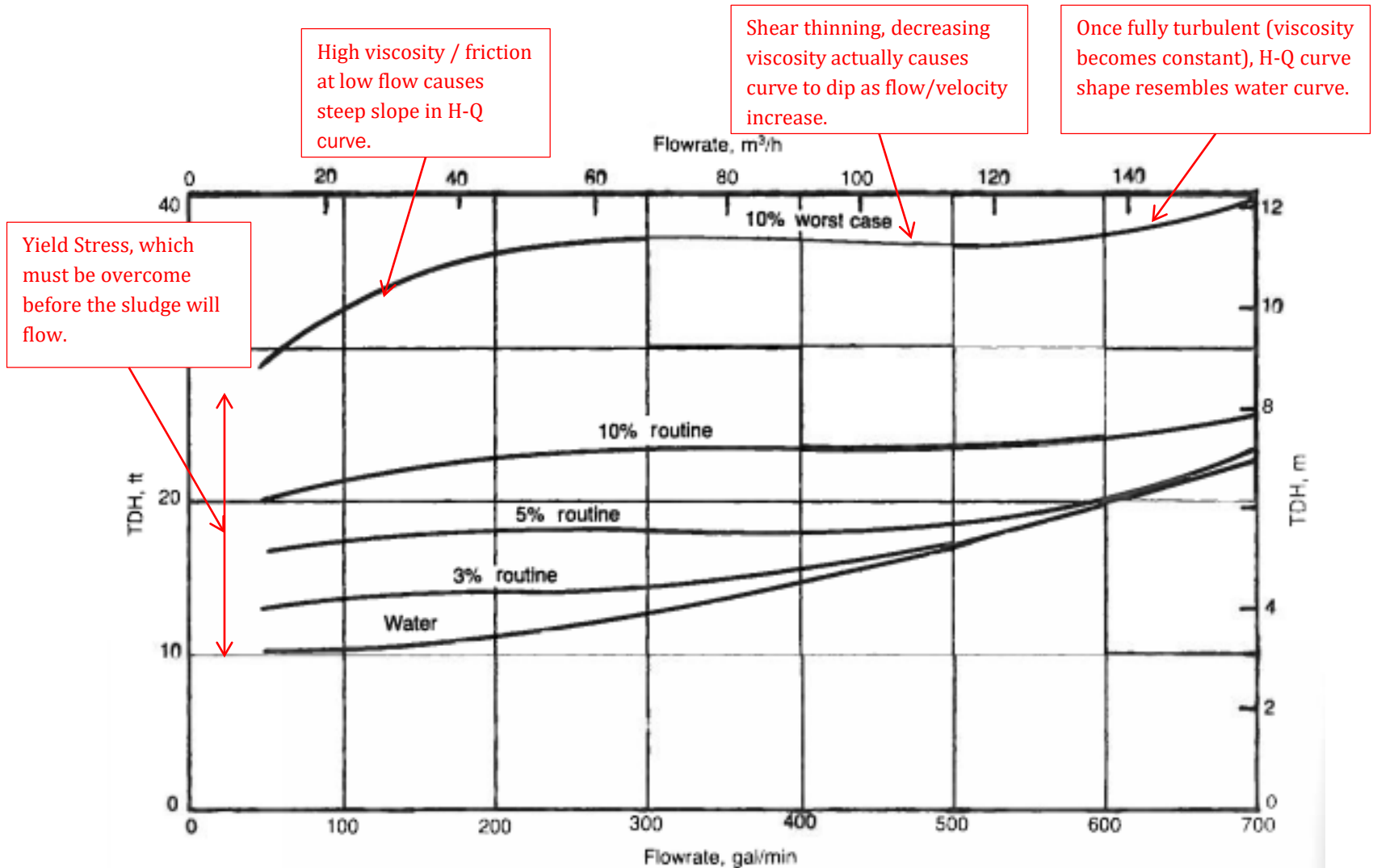
Fluid Types/Behaviors



What do we know about biosolids?

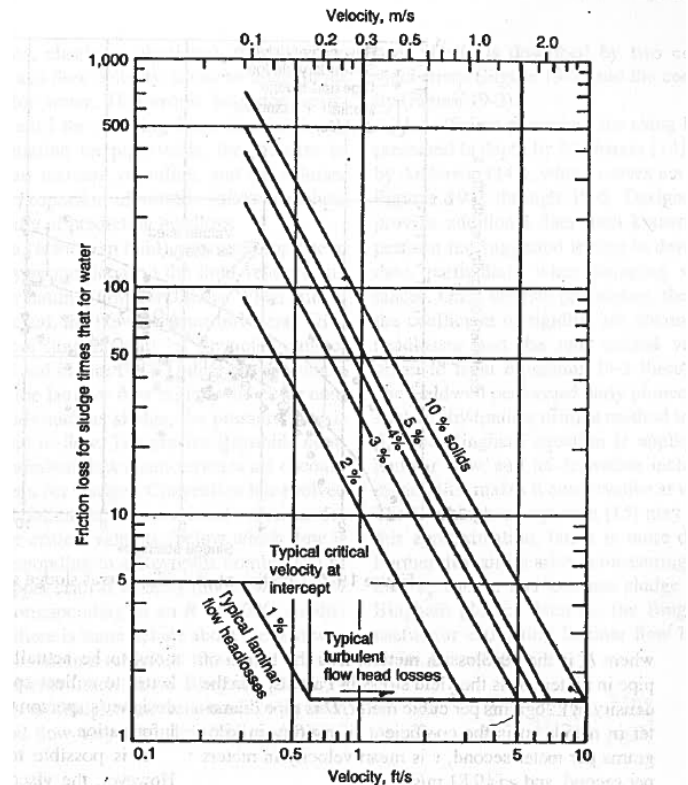
- Review of literature suggests digested sludge acts like a viscoplastic.
 - Shear-thinning
 - Yield stress

What does that look like on system curve?



Let's hit the books...

- Pumping Station Design (Sanks)
 - Underlying theory/equations
 - Pump types
 - Design procedures
- “Mulbargar Curves”



19-4. Recommended design curves. Headloss prediction for worst-case design.

- Acknowledges uncertainty in sludge friction loss calculations and recommends adjusting factors upward by “50% or more” for atypical sludges.

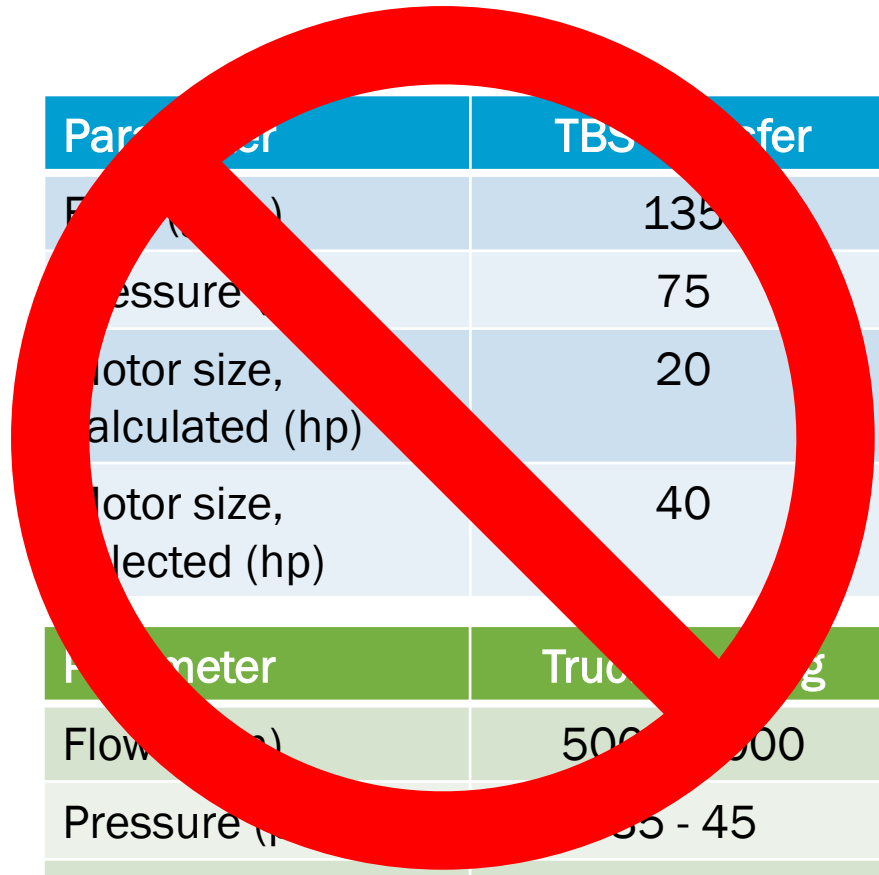
What did we come up with?

- Textbook – “Mulbargar” approach (based on 10% TS):

Parameter	TBS Transfer
Flow (gpm)	135
Pressure (psi)	75
Motor size, calculated (hp)	20
Motor size, selected (hp)	40

Parameter	Truck Loading
Flow (gpm)	500 - 1,000
Pressure (psi)	35 - 45
Motor size, calculated (hp)	60
Motor size, selected (hp)	125

So... we're done, right?



Parameter	TBS Transfer
Flow (gpm)	135
Pressure (psi)	75
Motor size, calculated (hp)	20
Motor size, selected (hp)	40

Parameter	TrueFlow
Flow (gpm)	500 - 600
Pressure (psi)	35 - 45

RESPECT THE RHEOLOGY!

Motor size, selected (hp)	125
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Too much uncertainty in sludge characteristics

- Two-pronged approach to improve our confidence in sludge pumping design:
 - Field testing
 - Rheology testing

Field Testing

- Centrifuge testing/optimization needed to produce 8-10% TS
- Existing thickened sludge pumps (rotary lobes) pumped to Digester 6 (digested sludge storage).
- Flow, pump speed, and manual pressure readings along the flow path. %TS grabs from centrifuges taken.



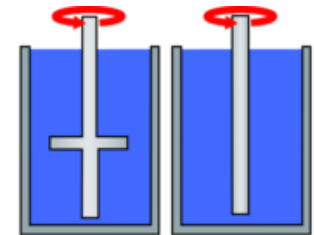
Field Testing - Findings

- Dialing in/maintaining at 9% or 10% is challenging.
- Existing TSPs were not going to be sufficient to pump all the way to the new storage tanks
- Data was noisy.
 - Pump speed, flow, and pressure loss weren't following clear relationship
- Estimated pressure loss roughly based on psi/ft.
 - ~0.05 psi/ft at 55-75 gpm

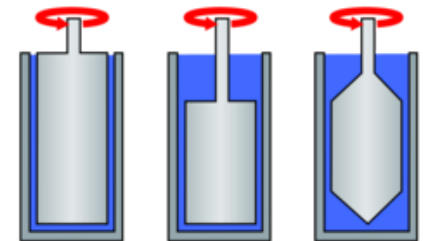
Rheology Analysis

- Field collected samples (at 9.6 %TS) sent to private lab
- Tested at two temperatures (40F and 75F)
- Tested over shear range of 0.01 to 100 s⁻¹
- STRESSTECH Rheometer (cup and bob)

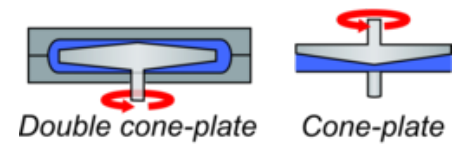
Rheometers



Spindle type



Concentric cylinder



Double cone-plate

Cone-plate

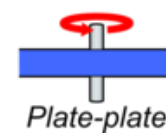
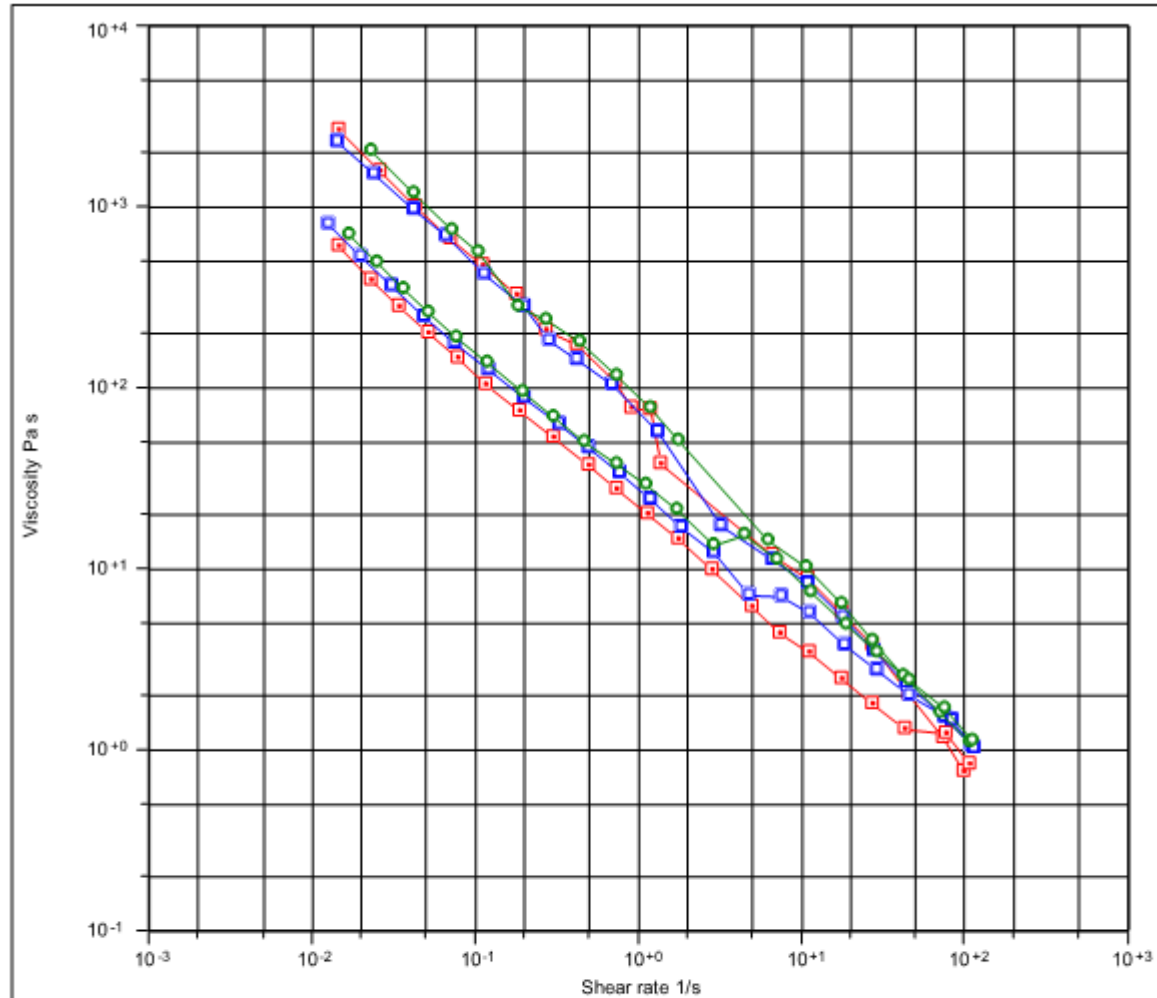


Plate-plate



Cone-cone

Shear Thinning Behavior



Thickened sludge 40F sh rate sweep r1.RSS

-□- Viscosity

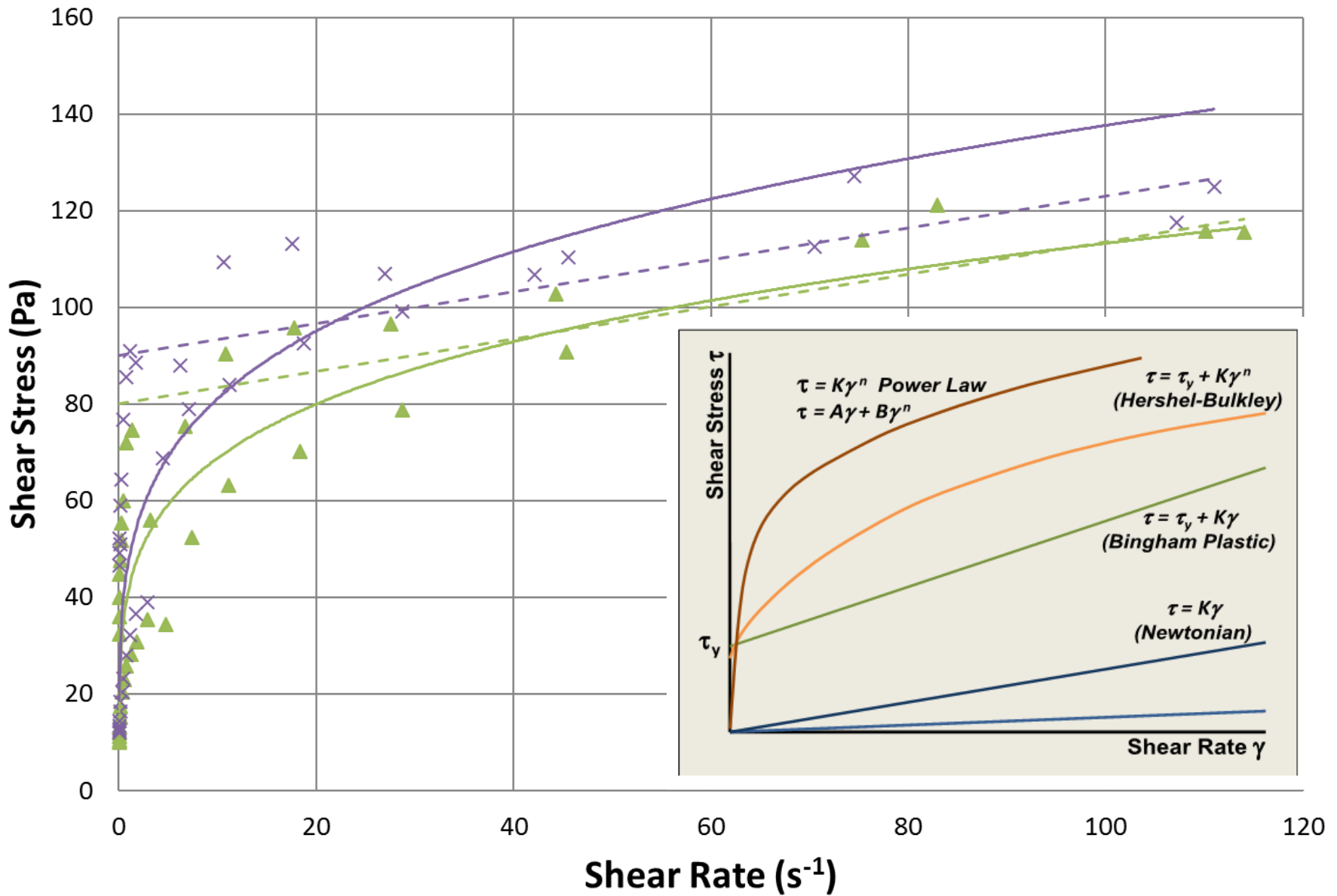
Thickened sludge 40F (Dup) sh rate sweep 1

-□- Viscosity

Thickened sludge 40F (Dup) sh rate sweep 2

-○- Viscosity

Two 40°F Runs with Power Law and Bingham Fits



Rheology Results: Constants

- Bingham:
 - Yield Stress, $\tau_y \sim 70$ to 90 Pa
 - Consistency Factor, $K \sim 0.15$ to 0.24 Pa-s
- Power Law:
 - Consistency Factor, $K \sim 35$ to 65 (Pa-s)ⁿ
 - Flow Index, $n \sim 0.13$ to 0.20

Avert your eyes... it's math

- Power Law – Laminar Case:

$$h_f = 4f \frac{L \bar{V}^2}{D} \quad \Delta p = \frac{2^{n+2} \left(\frac{3n+1}{n} \right)^n L K \bar{V}^n}{D^{n+1}}$$

$$f = \frac{2^{n+1} \left(\frac{3n+1}{n} \right)^n K}{\bar{V}^{2-n} D^n \rho} \quad Re_{PL} = 2^{3-n} \left(\frac{n}{3n+1} \right)^n \frac{\bar{V}^{2-n} D^n \rho}{K}$$
$$Re_{PL,critical} = 2100 \frac{(4n+2)(5n+3)}{3(3n+1)^2}$$

Hopefully you know someone with a spreadsheet...

TSP Headloss Pipe Diameter Changes.xlsxm - Excel

File Home Insert Page Layout Formulas Data Review View BLUEBEAM Tell me what you want to do

Clipboard Font Alignment Number Styles Cells

	A	B	C	D	E	F	G	H	I	J	K
1			TSP to BLAF								
2											
3		Segment #	1	2	3	4	5	6	7	8	
4		Segment Description	12" Suction	8" suction	8" Discharge	8" to 3" meter	3" meter	8" to 6"	6" to DCB3	8" to tank	
5		Flow (gpm) =	180	180	180	180	180	180	180	180	
6		Diameter (inch) =	12	8	8	8	3	8	6	8	
7		Length (ft) =	5	0	2	78	7	103	408	437	
8		Fitting K =	2.5	0.2	0.45	7.4		2	3.2	5.8	
9		Specific Gravity =	1.03	1.03	1.03	1.03	1.03	1.03	1.03	1.03	
10											
11		Consistency Factor, K (Pa.s^n) =	63.283	63.283	63.283	63.283	63.283	63.283	63.283	63.283	
12		Flow Index, n =	0.133	0.133	0.133	0.133	0.133	0.133	0.133	0.133	
13											
14	Turbulent Flow	Critical Reynolds Number =	1828	1828	1828	1828	1828	1828	1828	1828	
15		Reynolds Number =	2	10	10	10	339	10	28	10	
16	Est by WRC (UK)		Laminar Flow	Laminar Flow	Laminar Flow	Laminar Flow	Laminar Flow	Laminar Flow	Laminar Flow	Laminar Flow	
17		Velocity (m/s) =	0.156	0.350	0.350	0.350	2.490	0.350	0.623	0.350	
18		Fittings Factor =	870.6	202.9	202.9	202.9	6.9	202.9	72.7	202.9	
19											
20		D (m) =	0.30	0.20	0.20	0.20	0.08	0.20	0.15	0.20	
21		Q (m^3/sec) =	0.011	0.011	0.011	0.011	0.011	0.011	0.011	0.011	
22		L (m) =	2	0	1	24	2	31	124	133	
23		Density (kg/m^3) =	1030	1030	1030	1030	1030	1030	1030	1030	
24											
25		Wall Shear Stress (Pa) =	86.8	102.0	102.0	102.0	150.9	102.0	114.4	102.0	
26		Wall Shear Rate (1/s) =	10.7	36.3	36.3	36.3	687.5	36.3	85.9	36.3	
27											
28		Frictional Pressure (Pa) =	1736	0	1224	47745	16899	63048	373494	267495	

Updated Hydraulic Calculations - TBPs

Parameter	Mulbargar
Flow (gpm)	135
Pressure (psi)	75
Motor size, calculated (hp)	20
Motor size, selected (hp)	40



- Had to make sure piping class was OK

Updated Hydraulic Calculations – Truck Loading Pumps

Parameter	Mulbargar
Flow (gpm)	500 - 1,000
Pressure (psi)	35 - 45
Motor size, calculated (hp)	60
Motor size, selected (hp)	125

Startup in Oct 2016

- Loaded 50 trucks during operational demonstration
- Pressure readings on the TBPs pumping to BLAF tanks up to 200psi
- Pumping from centrifuges to BLAF regularly since startup
- Over 9 million gallons pumped in 2017 through Nov.

Project Takeaways/Lessons Learned

- No mixing in storage tanks – only circulation
 - At least 4 turn-overs before sampling begins to get representative sample
- %TS decreases over time when stored – continued digestion?
- Great feedback from the haulers on the loadout station!

Summary of Best Practices for Design

- Sanks lays it out pretty well:
 1. Treat each sludge pumping application as a unique design problem
 2. Develop site specific design criteria based on detailed evaluation of the specific sludge characteristics.
- Establish range of operating conditions from clean water to worst case sludge scenario – especially for centrifugal pumps.
- Common sludges like raw, or digested, less than 5-6% TS – “textbook” or simplified approaches likely OK.
- If data exists for a “similar” sludge use it with caution.
- Hydraulic modeling softwares come with sludge correction/rheology models – apply with engineering judgement (do some homework on limitations/applicability).

When to do Rheology Testing?

- Pumping design
 - Especially for really thick or unusual sludge characteristics where data isn't available.
- Mixing designs
 - Confirm how much energy is needed and if the sludge will mix

Thanks to everyone involved!

- Black and Veatch

- Bob O'Bryan
- Sierra McCreary
- Tyler York

- City of Columbus

- Troy Branson
- Rick Kent
- Everyone at SWWTP

- Brown and Caldwell

- Dave Nitz
- Dante Fiorino
- Ravi Ravisangar (the man with the spreadsheet!)

Questions

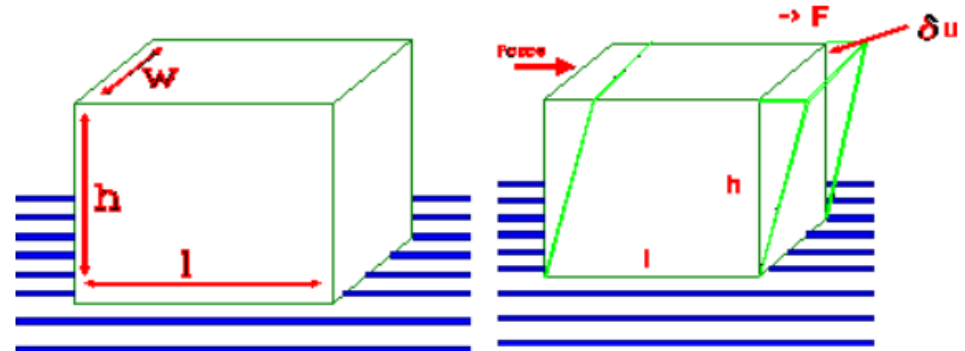


Bullpen



More Rheology

- Shear Strain = $\delta u/h$
- Shear Stress = $F/A = \tau = \text{N/m}^2$
- Shear Rate (Shear Strain Rate) = $\delta \cdot \text{Shear Strain} / \delta \cdot \text{time} = \text{s}^{-1}$
- Viscosity (dynamic or absolute) = Shear Stress/Shear Rate = Ns/m^2 (Pa-s or Poise)
- Kinematic (divide by density) = m^2/s (Stokes)



Pump Types

- Centrifugal- Non-Clog
- Centrifugal- Recessed Impeller
- Screw-Centrifugal
- Progressive Cavity
- Rotary Lobe
- Piston Pumps
- Diaphragm